Boundary Layer Solutions

1 Important BL Parameters

1.1 99% BL Thickness

 δ_{99} is the distance from wall such that $u = 0.99U_{\infty}$.

1.2 Displacement Thickness

Based on mass conservation, mass flow deficit due to $u < U_{\infty}$ in decelerated flow in boundary layer is:

$$\rho \int U_{\infty} \, dy - \rho \int u \, dy = \rho \int (U_{\infty} - u) \, dy \tag{1}$$

We define δ^* as:

$$\delta^* = \int_0^\infty (1 - \frac{u}{U_\infty}) \, dy \tag{2}$$

1.3 Momentum Thickness

Based on momentum conservation, reduced mass flow also implies less momentum.

$$\rho U_{\infty}^2 \theta = \int_0^\infty u (U_{\infty} - u) \, dy \tag{3}$$

$$\theta = \int_0^\infty \frac{u}{U_\infty} (1 - \frac{u}{U_\infty}) \, dy \tag{4}$$

1.4 Wall Shear Stress

For Newtonian fluid and impermeable wall (v = 0 for all x):

$$\tau_w = \mu \frac{\partial u}{\partial y}|_{y=0} \tag{5}$$

1.5 Friction Coefficient

$$C_f = \frac{\tau_w}{\frac{1}{2}\rho U_\infty^2} \tag{6}$$

2 Boundary Layer Transformations (Simplified)

Assume a 2D stream function as:

$$\psi(x,y) = g(x)f(\eta) \tag{7}$$

 η is to be non-dimensional, same as $f(\eta)$, which means g(x) has the dimensions $[L^2][T^{-1}]$. Then, we define η as non-dimensional distance:

$$\eta = \frac{y}{\frac{g(x)}{U_{\infty}}} = \frac{U_{\infty}y}{g(x)}$$
(8)

Based on the definition of stream function:

$$u = \frac{\partial \psi}{\partial y} \tag{9}$$

$$v = -\frac{\partial \psi}{\partial x} \tag{10}$$

Plug into equation:

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} = U_{\infty}\frac{dU_{\infty}}{dx} + \nu\frac{\partial^2 u}{\partial y^2}$$
(11)

$$\frac{\partial \psi}{\partial y} \frac{\partial^2 \psi}{\partial x \partial y} - \frac{\partial \psi}{\partial x} \frac{\partial^2 \psi}{\partial y^2} = U_{\infty} \frac{dU_{\infty}}{dx} + \nu \frac{\partial^3 \psi}{\partial y^3}$$

Then apply the transformation: e.g.

$$\frac{\partial \psi}{\partial y} = g(x)\frac{\partial f}{\partial y} = g(x)f'\frac{\partial \eta}{\partial y} = U_{\infty}f'$$

where primes are derivatives in η , and we have used $\partial \eta / \partial y = U_{\infty}/g$. For derivatives in x, using the quotient rule of calculus, we get

$$\frac{\partial \eta}{\partial x} = y \frac{d}{dx} \left(\frac{U_{\infty}}{g} \right) = y \frac{g dU_{\infty}/dx - U_{\infty} dg/dx}{g^2}$$

With $\eta = U_{\infty}y/g$, to express stuff in terms of η , as far as possible:

$$\frac{\partial \eta}{\partial x} = \eta \left[\frac{1}{U_{\infty}} \frac{dU_{\infty}}{dx} - \frac{1}{g} \frac{dg}{dx} \right]$$

The math does get longer yet:

$$\frac{\partial^2 \psi}{\partial x \partial y} = \frac{\partial}{\partial x} \left[\frac{\partial \psi}{\partial y} \right] = \frac{\partial}{\partial x} \left[U_{\infty} f' \right] = U_{\infty} f'' \frac{\partial \eta}{\partial x} + f' \frac{dU_{\infty}}{dx}$$

After some **magical math**, we get:

$$f''' + \frac{g}{\nu U_{\infty}} \frac{dg}{dx} f f'' + \frac{g^2}{\nu U_{\infty}^2} \frac{dU_{\infty}}{dx} (1 - f'^2) = 0$$
(12)

Here, we define:

$$\alpha = \frac{g}{\nu U_{\infty}} \frac{dg}{dx} \tag{13}$$

$$\beta = \frac{g^2}{\nu U_\infty^2} \frac{dU_\infty}{dx} \tag{14}$$

To make sure f and its derivatives are all non-dimensional, both α and β should be non-dimensional constants. A different value of α simply changes the definition of g(x)by a constant factor. Then, without loss of generality, we take $\alpha = 1$.

$$\frac{d}{dx}\left(\frac{g^2}{U_{\infty}}\right) = \frac{1}{U_{\infty}}\frac{dg^2}{dx} - \frac{g^2}{U_{\infty}^2}\frac{dU_{\infty}}{dx} = (2-\beta)\nu \tag{15}$$

Integrating in x, where x is measured from the leading edge, we get:

$$g^2 = (2 - \beta)\nu U_{\infty} x \tag{16}$$

$$\beta = \frac{(2-\beta)\nu U_{\infty} x}{U_{\infty}^2 \nu} \frac{dU_{\infty}}{dx}$$
(17)

Or:

$$\frac{1}{U_{\infty}}\frac{dU_{\infty}}{dx} = \left(\frac{\beta}{2-\beta}\right)\left(\frac{1}{x}\right) \tag{18}$$

Integration on both sides will give:

$$U_{\infty}(x) = Cx^m \tag{19}$$

Where C is a constant, and

$$m = \frac{\beta}{2 - \beta} \tag{20}$$

$$\beta = \frac{2m}{m+1} \tag{21}$$

3 Blasius Solution

3.1 Derivation

Based on the power law, m = 0 means the boundary layer over a flat plate, with zero angle of attack. This also implies $\beta = 0$, then:

$$g = \sqrt{2\nu U_{\infty} x} \tag{22}$$

$$\eta = \frac{U_{\infty}y}{\sqrt{2\nu U_{\infty}x}} = \frac{y}{\sqrt{2\nu x/U_{\infty}}}$$
(23)

Finally it reduces to:

$$f^{'''} + ff^{''} = 0 \tag{24}$$

Check the Blasius table to get the solution.

3.2 Velocity

$$u = U_{\infty} f' \tag{25}$$

$$v = \frac{1}{2}\sqrt{\frac{\nu U_{\infty}}{x}}(\eta f' - f) \tag{26}$$

3.3 Thickness

$$\delta_{99} = 5\sqrt{\frac{\nu x}{U_{\infty}}} \tag{27}$$

$$\delta^* = \sqrt{\frac{\nu x}{U_{\infty}}} \int_0^\infty 1 - f' \, d\eta = 1.7208 \sqrt{\frac{\nu x}{U_{\infty}}} \tag{28}$$

$$\theta(x) = 0.664 \sqrt{\frac{\nu x}{U_{\infty}}} \tag{29}$$

3.4 Wall Shear Stress and Friction Coefficient

$$\tau_w = \mu U_\infty f''(0) \sqrt{\frac{\nu x}{U_\infty}} \tag{30}$$

$$C_f(x) = 2f''(0)\sqrt{\frac{\nu}{U_{\infty}x}} = 0.664Re_x^{-1/2}$$
(31)

4 Other Solutions

Power Law:

$$U_{\infty} = Ax^m \tag{32}$$

4.1 m = 0

Blasius Solution.

4.2 m = 1

Stagnation flow, but in the region far from the stagnation point.

4.3 m greater than 0

Accelerating boundary layer, with negative (favorable) pressure gradient.

$$\frac{1}{\rho}\frac{dP}{dx} = -U_{\infty}\frac{dU_{\infty}}{dx} = -A^2x^m mx^{m-1} < 0 \tag{33}$$

4.4 m smaller than 0

Decelerating boundary layer, with positive (adverse) pressure gradient. Separation can occur.